

HEAT TRANSFER DATA SHEETS

$$\text{Nusselt number} = \text{Nu} = \frac{hL}{k}$$

$$\text{Sherwood number} = \text{Sh} = \frac{h_m L}{D}$$

$$\text{Grashof number} = \text{Gr} = \frac{g \beta \Delta T \rho^2 L^3}{\mu^2} = \frac{g \beta \Delta T L^3}{\nu^2}$$

$$\text{Prandtl number} = \text{Pr} = \frac{C\mu}{k} = \frac{\nu}{\alpha}$$

$$\text{Schmidt number} = \text{Sc} = \frac{\nu}{D}$$

$$\text{Rayleigh number} = \text{Ra} = \text{Gr} \times \text{Pr}$$

$$\beta = \frac{1}{T} \text{ for ideal gas}$$

$$\frac{1}{UA} = \left(\frac{1}{h_i A_i} + \frac{F_{f,i}}{A_i} + \frac{\ell \ln\left(\frac{r_2}{r_1}\right)}{2\pi k L} + \frac{F_{f,o}}{A_o} + \frac{1}{h_o A_o} \right)$$

$$\Delta T_{lm} = \frac{\Delta T_2 - \Delta T_1}{\ln(\Delta T_2 / \Delta T_1)}$$

$$\text{NTU} = \frac{UA}{C_{\min}}$$

$$\varepsilon = \frac{C_h(T_{hi} - T_{ho})}{C_{\min}(T_{hi} - T_{ci})} = \frac{C_c(T_{co} - T_{ci})}{C_{\min}(T_{hi} - T_{ci})}$$

CONVECTION HEAT TRANSFER CORRELATIONS

1. External Flows

Local Nu for laminar flow on a flat plate with constant surface temperature. $0.6 \leq Pr \leq 50$.

$$Nu_x = 0.332 Pr^{1/3} Re_x^{1/2}$$

Average Nu for laminar flow over flat plate with constant surface temperature. $0.6 \leq Pr \leq 50$.

$$\overline{Nu_x} = 0.664 Pr^{1/3} Re_x^{1/2}$$

Local Nu for laminar flow on a flat plate with constant heat flux. $0.6 \leq Pr \leq 50$.

$$Nu_x = 0.453 Pr^{1/3} Re_x^{1/2}$$

Local Nu for turbulent flow over a flat plate with constant surface temperature.
 $5 \times 10^5 \leq Re_x \leq 10^8$; $0.6 \leq Pr \leq 60$

$$Nu_x = 0.0296 Re_x^{0.8} Pr^{1/3}$$

Average Nu for turbulent flow over a flat plate with constant surface temperature.
 $5 \times 10^5 \leq Re_x \leq 10^8$; $0.6 \leq Pr \leq 60$

$$\overline{Nu_L} = 0.037 Re_L^{0.8} Pr^{1/3}$$

Average Nu for mixed flow conditions over flat plate, i.e. starting laminar and becoming turbulent.
 $Re_{x,crit} = 5 \times 10^5$; $Re_L < 10^8$; $0.6 < Pr < 60$.

$$\overline{Nu_L} = Pr^{1/3} (0.037 Re_L^{0.8} - 870)$$

Average Nu for cylinder in cross flow. $Pr \geq 0.7$

$$\overline{Nu_D} = C Re_D^m Pr^{1/3}$$

Re_D	C	m
0.4 - 4	0.989	0.330
4 - 40	0.911	0.385
40 - 4000	0.683	0.466
4000 - 40,000	0.193	0.618
40,000 - 400,000	0.027	0.805

2. Internal flow in Circular Channels

Laminar fully developed flow with constant heat flux. $Pr \geq 0.6$

$$Nu_D = 4.36$$

Laminar fully developed flow with constant surface temperature. $Pr \geq 0.6$

$$Nu_D = 3.66$$

Laminar, combined entry length with constant surface temperature

$$Nu_D = 1.86 \left(\frac{Re_D Pr}{L/D} \right)^{1/3}$$

Fully developed turbulent flow. $0.6 \leq Pr \leq 160$; $Re_D \geq 2500$; $L/D \geq 10$; $n = 0.4$ for heated tube and 0.3 for cooled tube.

$$Nu_D = 0.023 Re_D^{4/5} Pr^n$$

Not fully developed turbulent flow (entrance region). $10 \leq L/D \leq 400$ (Properties at mean bulk temperature).

$$Nu_D = 0.036 Re^{0.8} Pr^{1/3} (D/L)^{0.055}$$

3. Free Convection

Flat vertical plate with constant surface temperature, all values of Ra_L

$$\overline{Nu}_L = \left[0.825 + \frac{0.387 Ra_L^{1/6}}{\left[1 + (0.492 / Pr)^{9/16} \right]^{8/27}} \right]^2$$

Horizontal plate hot surface up or cold surface down. Characteristic length = surface area ÷ perimeter. $10^4 \leq Ra_L \leq 10^7$; and $10^7 \leq Ra_L \leq 10^{11}$ respectively.

$$\overline{Nu}_L = 0.54 Ra_L^{1/4}$$

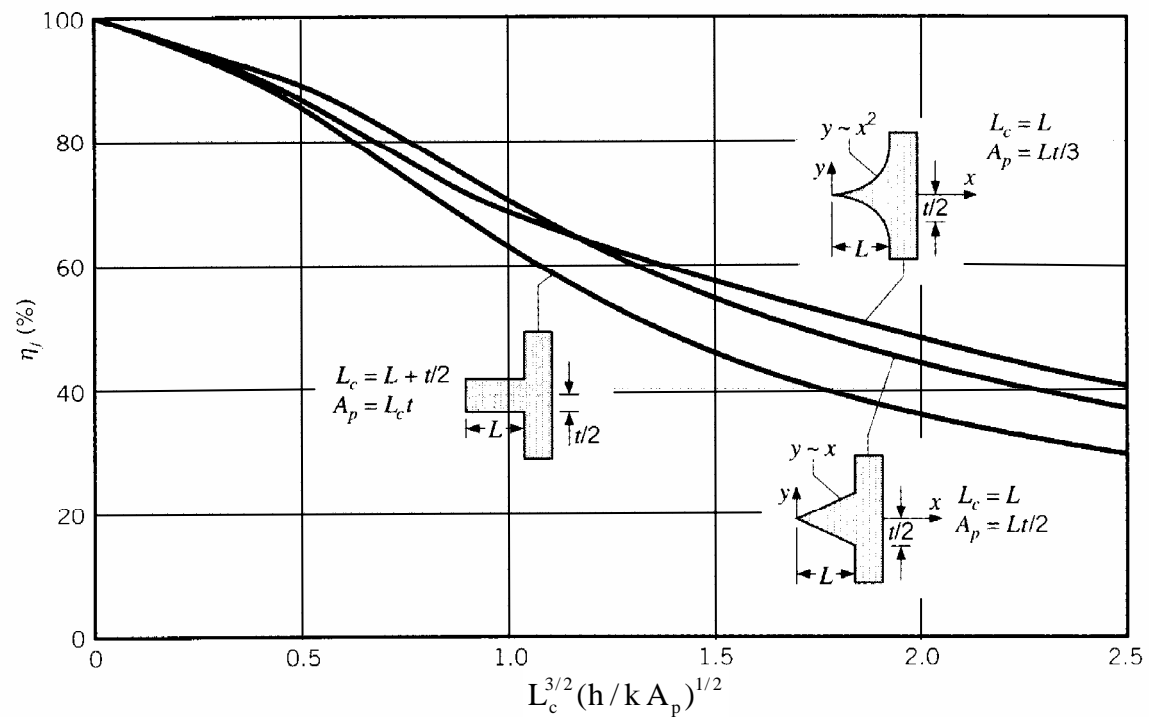
$$\overline{Nu}_L = 0.15 Ra_L^{1/3}$$

Flat plate cold surface up or hot surface down. $10^5 \leq Ra_L \leq 10^{10}$. Characteristic length = surface area \div perimeter.

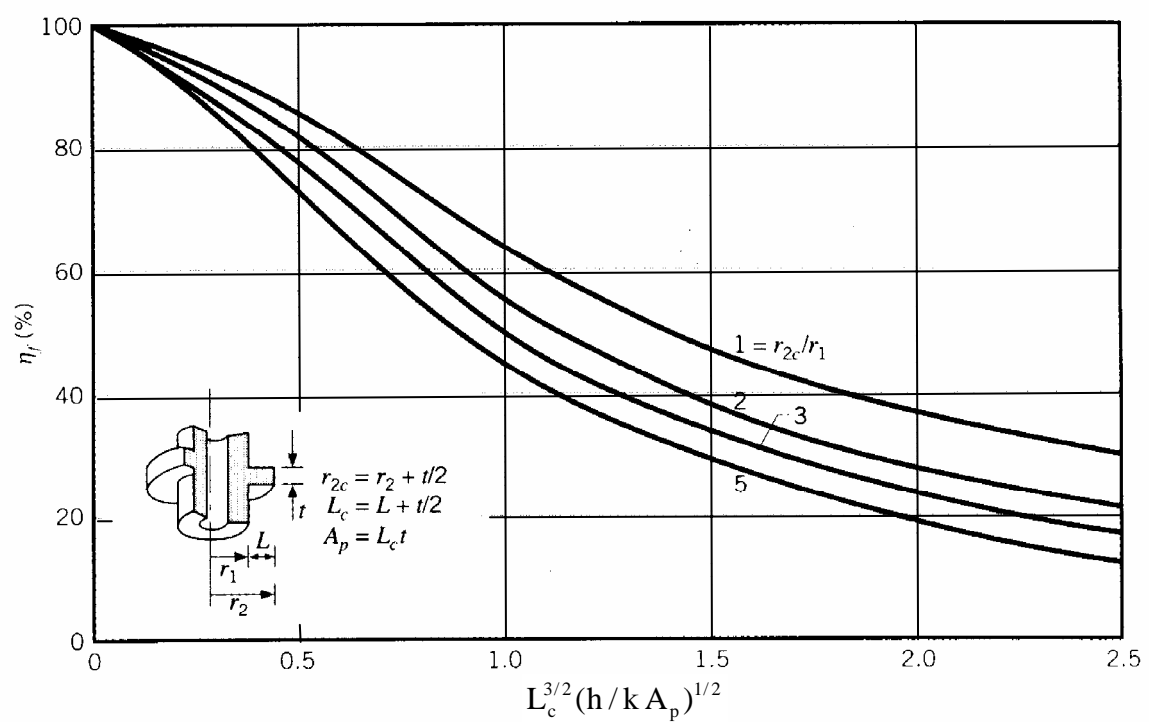
$$\overline{Nu}_L = 0.27 Ra_L^{1/4}$$

Horizontal cylinder. $Ra_D \leq 10^{12}$

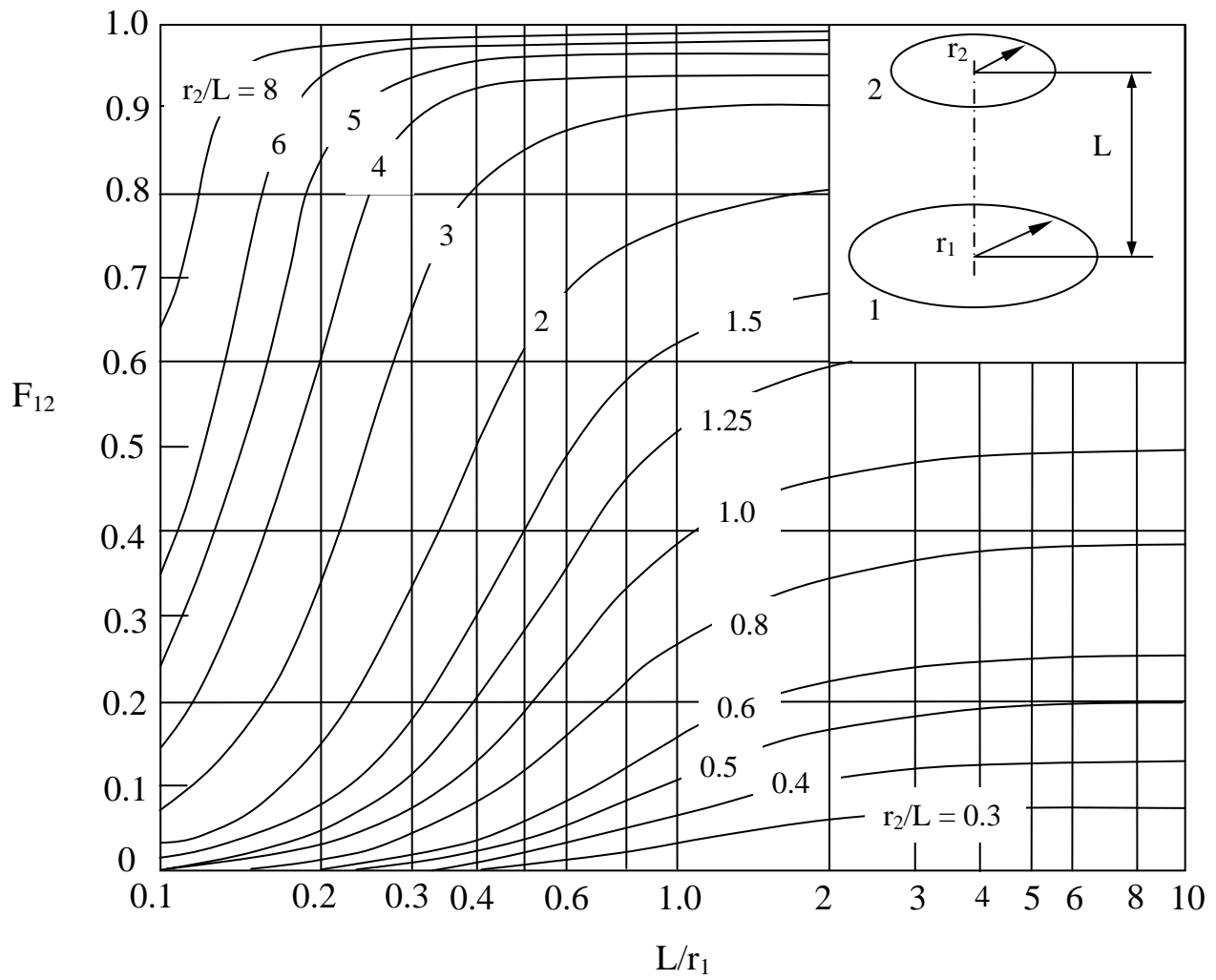
$$\overline{Nu}_D = \left[0.6 + \frac{0.387 Ra_D^{1/6}}{\left[1 + (0.559 / Pr)^{9/16} \right]^{8/27}} \right]^2$$



Efficiency of straight fins (rectangular, triangular and parabolic profile)



Efficiency of annular fins of rectangular profile



View Factor for two coaxial parallel discs

Saturated Water and Steam

t [°C]	p_s [bar]	v_f $10^{-2}[\text{m}^3/\text{kg}]$	c_{pf} c_{pg} [kJ/kg K]	μ_f μ_g $10^{-6}[\text{kg}/\text{m s}]$	k_f k_g $10^{-6}[\text{kW}/\text{m K}]$	$(Pr)_f$	$(Pr)_g$
0.01	0.006112	0.10002	4.210 1.86	1752 8.49	569 16.3	12.96	0.97
5	0.008719	0.10001	4.204 1.86	1501 8.66	578 16.7	10.92	0.96
10	0.01227	0.10003	4.193 1.86	1300 8.83	587 17.1	9.29	0.96
15	0.01704	0.10010	4.186 1.87	1136 9.00	595 17.5	7.99	0.96
20	0.02337	0.10018	4.183 1.87	1002 9.18	603 17.9	6.95	0.96
25	0.03166	0.10030	4.181 1.88	890 9.35	611 18.3	6.09	0.96
30	0.04242	0.10044	4.179 1.88	797 9.52	618 18.7	5.39	0.96
35	0.05622	0.10060	4.178 1.88	718 9.70	625 19.1	4.80	0.96
40	0.07375	0.10079	4.179 1.89	651 9.87	632 19.5	4.30	0.96
45	0.09582	0.10099	4.181 1.89	594 10.0	638 19.9	3.89	0.95
50	0.1233	0.1012	4.182 1.90	544 10.2	643 20.4	3.54	0.95
55	0.1574	0.1015	4.183 1.90	501 10.4	648 20.8	3.23	0.95
60	0.1992	0.1017	4.185 1.91	463 10.6	653 21.2	2.97	0.95
65	0.2501	0.1020	4.188 1.92	430 10.7	658 21.6	2.74	0.95
70	0.3116	0.1023	4.191 1.93	400 10.9	662 22.0	2.53	0.96
75	0.3855	0.1026	4.194 1.94	374 11.1	666 22.5	2.36	0.96
80	0.4736	0.1029	4.198 1.95	351 11.3	670 22.9	2.20	0.96
85	0.5780	0.1032	4.203 1.96	330 11.4	673 23.3	2.06	0.96
90	0.7011	0.1036	4.208 1.97	311 11.6	676 23.8	1.94	0.96
95	0.8453	0.1040	4.213 1.99	294 11.8	678 24.3	1.83	0.97
100	1.01325	0.1044	4.219 2.01	279 12.0	681 24.8	1.73	0.97
105	1.208	0.1048	4.226 2.03	265 12.2	683 25.3	1.64	0.98
110	1.433	0.1052	4.233 2.05	252 12.4	684 25.8	1.56	0.99
115	1.691	0.1056	4.240 2.07	241 12.6	686 26.3	1.49	0.99
120	1.985	0.1060	4.248 2.09	230 12.8	687 26.8	1.42	1.00
125	2.321	0.1065	4.26 2.12	220 13.0	687 27.3	1.36	1.01
130	2.701	0.1070	4.27 2.15	211 13.2	688 27.8	1.31	1.02
135	3.131	0.1075	4.28 2.18	203 13.4	688 28.3	1.26	1.03
140	3.614	0.1080	4.29 2.21	195 13.5	688 28.8	1.22	1.04
145	4.155	0.1085	4.30 2.25	188 13.7	687 29.4	1.18	1.05
150	4.760	0.1091	4.32 2.29	181 13.9	687 30.0	1.14	1.07
160	6.181	0.1102	4.35 2.38	169 14.2	684 31.3	1.07	1.09
170	7.920	0.1114	4.38 2.49	159 14.6	681 32.6	1.02	1.12
180	10.03	0.1128	4.42 2.62	149 15.0	676 34.1	0.97	1.15
190	12.55	0.1142	4.46 2.76	141 15.3	671 35.7	0.94	1.18
200	15.55	0.1157	4.51 2.91	134 15.7	665 37.5	0.91	1.22
210	19.08	0.1173	4.56 3.07	127 16.0	657 39.4	0.88	1.25
220	23.20	0.1190	4.63 3.25	121 16.3	648 41.5	0.86	1.28
230	27.98	0.1209	4.70 3.45	116 16.7	639 43.9	0.85	1.31
240	33.48	0.1229	4.78 3.68	111 17.1	628 46.5	0.84	1.35
250	39.78	0.1251	4.87 3.94	107 17.5	616 49.5	0.85	1.39
260	46.94	0.1276	4.98 4.22	103 17.9	603 52.8	0.85	1.43
270	55.05	0.1302	5.10 4.55	99 18.3	589 56.6	0.86	1.47
280	64.19	0.1332	5.24 4.98	96 18.8	574 61.0	0.88	1.53
290	74.45	0.1366	5.42 5.46	93 19.3	558 66.0	0.90	1.60
300	85.92	0.1404	5.65 6.18	90 19.8	541 72.0	0.94	1.70
320	112.9	0.1499					
340	146.1	0.1639					
360	186.7	0.1894					
370	210.5	0.2225					
374.15	221.2	0.317					

The values for saturated water can be used with good accuracy above saturation pressure. The values for saturated steam can be used with only moderate accuracy below saturation pressure at temperatures greater than 200 °C.

Saturated Water and Steam

t [°C]	p_s [bar]	v_g [m ³ /kg]	h_f	h_{fg} [kJ/kg]	h_g	s_f	s_{fg} [kJ/kg K]	s_g
0.01	0.006112	206.1	0*	2500.8	2500.8	0†	9.155	9.155
1	0.006566	192.6	4.2	2498.3	2502.5	0.015	9.113	9.128
2	0.007054	179.9	8.4	2495.9	2504.3	0.031	9.071	9.102
3	0.007575	168.2	12.6	2493.6	2506.2	0.046	9.030	9.076
4	0.008129	157.3	16.8	2491.3	2508.1	0.061	8.989	9.050
5	0.008719	147.1	21.0	2488.9	2509.9	0.076	8.948	9.024
6	0.009346	137.8	25.2	2486.6	2511.8	0.091	8.908	8.999
7	0.01001	129.1	29.4	2484.3	2513.7	0.106	8.868	8.974
8	0.01072	121.0	33.6	2481.9	2515.5	0.121	8.828	8.949
9	0.01147	113.4	37.8	2479.6	2517.4	0.136	8.788	8.924
10	0.01227	106.4	42.0	2477.2	2519.2	0.151	8.749	8.900
11	0.01312	99.90	46.2	2474.9	2521.1	0.166	8.710	8.876
12	0.01401	93.83	50.4	2472.5	2522.9	0.180	8.671	8.851
13	0.01497	88.17	54.6	2470.2	2524.8	0.195	8.633	8.828
14	0.01597	82.89	58.8	2467.8	2526.6	0.210	8.594	8.804
15	0.01704	77.97	62.9	2465.5	2528.4	0.224	8.556	8.780
16	0.01817	73.38	67.1	2463.1	2530.2	0.239	8.518	8.757
17	0.01936	69.09	71.3	2460.8	2532.1	0.253	8.481	8.734
18	0.02063	65.08	75.5	2458.4	2533.9	0.268	8.444	8.712
19	0.02196	61.34	79.7	2456.0	2535.7	0.282	8.407	8.689
20	0.02337	57.84	83.9	2453.7	2537.6	0.296	8.370	8.666
21	0.02486	54.56	88.0	2451.4	2539.4	0.310	8.334	8.644
22	0.02642	51.49	92.2	2449.0	2541.2	0.325	8.297	8.622
23	0.02808	48.62	96.4	2446.6	2543.0	0.339	8.261	8.600
24	0.02982	45.92	100.6	2444.2	2544.8	0.353	8.226	8.579
25	0.03166	43.40	104.8	2441.8	2546.6	0.367	8.190	8.557
26	0.03360	41.03	108.9	2439.5	2548.4	0.381	8.155	8.536
27	0.03564	38.81	113.1	2437.2	2550.3	0.395	8.120	8.515
28	0.03778	36.73	117.3	2434.8	2552.1	0.409	8.085	8.494
29	0.04004	34.77	121.5	2432.4	2553.9	0.423	8.050	8.473
30	0.04242	32.93	125.7	2430.0	2555.7	0.436	8.016	8.452
32	0.04754	29.57	134.0	2425.3	2559.3	0.464	7.948	8.412
34	0.05318	26.60	142.4	2420.5	2562.9	0.491	7.881	8.372
36	0.05940	23.97	150.7	2415.8	2566.5	0.518	7.814	8.332
38	0.06624	21.63	159.1	2411.0	2570.1	0.545	7.749	8.294
40	0.07375	19.55	167.5	2406.2	2573.7	0.572	7.684	8.256
42	0.08198	17.69	175.8	2401.4	2577.2	0.599	7.620	8.219
44	0.09100	16.03	184.2	2396.6	2580.8	0.625	7.557	8.182
46	0.1009	14.56	192.5	2391.8	2584.3	0.651	7.494	8.145
48	0.1116	13.23	200.9	2387.0	2587.9	0.678	7.433	8.111
50	0.1233	12.04	209.3	2382.1	2591.4	0.704	7.371	8.075
55	0.1574	9.578	230.2	2370.1	2600.3	0.768	7.223	7.991
60	0.1992	7.678	251.1	2357.9	2609.0	0.831	7.078	7.909
65	0.2501	6.201	272.0	2345.7	2617.7	0.893	6.937	7.830
70	0.3116	5.045	293.0	2333.3	2626.3	0.955	6.800	7.755
75	0.3855	4.133	313.9	2320.8	2634.7	1.015	6.666	7.681
80	0.4736	3.408	334.9	2308.3	2643.2	1.075	6.536	7.611
85	0.5780	2.828	355.9	2295.6	2651.5	1.134	6.410	7.544
90	0.7011	2.361	376.9	2282.8	2659.7	1.192	6.286	7.478
95	0.8453	1.982	398.0	2269.8	2667.8	1.250	6.166	7.416
100	1.01325	1.673	419.1	2256.7	2675.8	1.307	6.048	7.355

† u and s are chosen to be zero for saturated liquid at the triple point.

Note: values of v_f can be found on p. 10.

Dry Air at Low Pressure

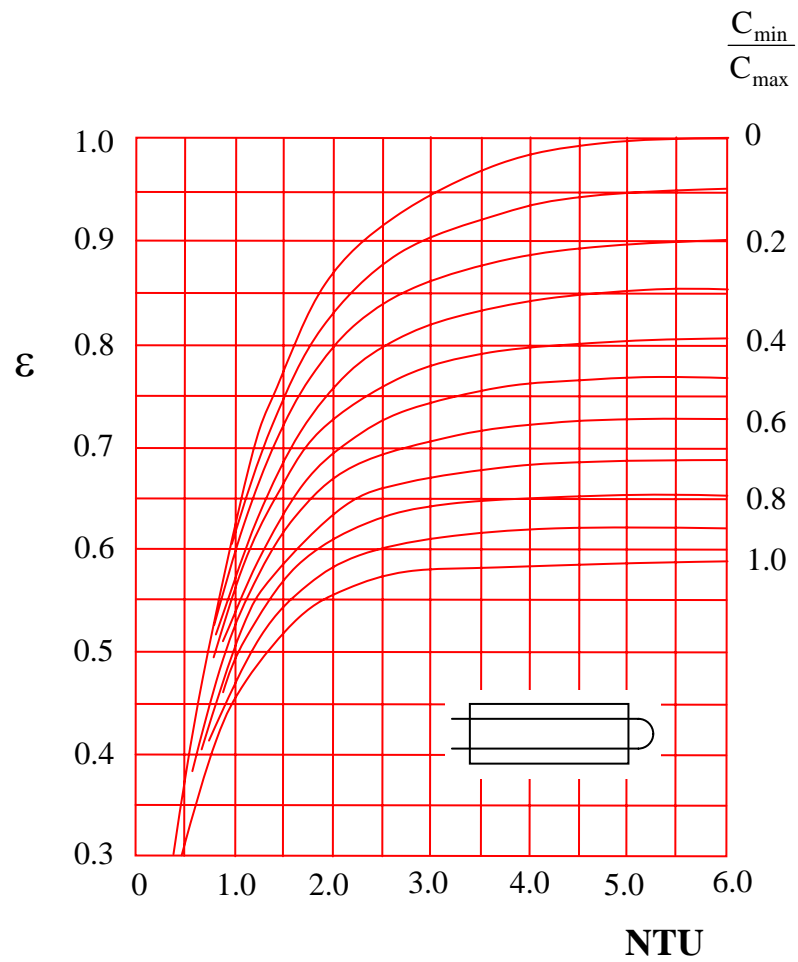
T [K]							at 1 atm	
	c_p [kJ/kg K]	c_v	γ	μ 10^{-5} [kg/m s]	k 10^{-5} [kW/m K]	Pr	ρ [kg/m ³]	ν 10^{-5} [m ² /s]
175	1.0023	0.7152	1.401	1.182	1.593	0.744	2.017	0.586
200	1.0025	0.7154	1.401	1.329	1.809	0.736	1.765	0.753
225	1.0027	0.7156	1.401	1.467	2.020	0.728	1.569	0.935
250	1.0031	0.7160	1.401	1.599	2.227	0.720	1.412	1.132
275	1.0038	0.7167	1.401	1.725	2.428	0.713	1.284	1.343
300	1.0049	0.7178	1.400	1.846	2.624	0.707	1.177	1.568
325	1.0063	0.7192	1.400	1.962	2.816	0.701	1.086	1.807
350	1.0082	0.7211	1.398	2.075	3.003	0.697	1.009	2.056
375	1.0106	0.7235	1.397	2.181	3.186	0.692	0.9413	2.317
400	1.0135	0.7264	1.395	2.286	3.365	0.688	0.8824	2.591
450	1.0206	0.7335	1.391	2.485	3.710	0.684	0.7844	3.168
500	1.0295	0.7424	1.387	2.670	4.041	0.680	0.7060	3.782
550	1.0398	0.7527	1.381	2.849	4.357	0.680	0.6418	4.439
600	1.0511	0.7640	1.376	3.017	4.661	0.680	0.5883	5.128
650	1.0629	0.7758	1.370	3.178	4.954	0.682	0.5430	5.853
700	1.0750	0.7879	1.364	3.332	5.236	0.684	0.5043	6.607
750	1.0870	0.7999	1.359	3.482	5.509	0.687	0.4706	7.399
800	1.0987	0.8116	1.354	3.624	5.774	0.690	0.4412	8.214
850	1.1101	0.8230	1.349	3.763	6.030	0.693	0.4153	9.061
900	1.1209	0.8338	1.344	3.897	6.276	0.696	0.3922	9.936
950	1.1313	0.8442	1.340	4.026	6.520	0.699	0.3716	10.83
1000	1.1411	0.8540	1.336	4.153	6.754	0.702	0.3530	11.76
1050	1.1502	0.8631	1.333	4.276	6.985	0.704	0.3362	12.72
1100	1.1589	0.8718	1.329	4.396	7.209	0.707	0.3209	13.70
1150	1.1670	0.8799	1.326	4.511	7.427	0.709	0.3069	14.70
1200	1.1746	0.8875	1.323	4.626	7.640	0.711	0.2941	15.73
1250	1.1817	0.8946	1.321	4.736	7.849	0.713	0.2824	16.77
1300	1.1884	0.9013	1.319	4.846	8.054	0.715	0.2715	17.85
1350	1.1946	0.9075	1.316	4.952	8.253	0.717	0.2615	18.94
1400	1.2005	0.9134	1.314	5.057	8.450	0.719	0.2521	20.06
1500	1.2112	0.9241	1.311	5.264	8.831	0.722	0.2353	22.36
1600	1.2207	0.9336	1.308	5.457	9.199	0.724	0.2206	24.74
1700	1.2293	0.9422	1.305	5.646	9.554	0.726	0.2076	27.20
1800	1.2370	0.9499	1.302	5.829	9.899	0.728	0.1961	29.72
1900	1.2440	0.9569	1.300	6.008	10.233	0.730	0.1858	32.34
2000	1.2505	0.9634	1.298	—	—	—	0.1765	—
2100	1.2564	0.9693	1.296	—	—	—	0.1681	—
2200	1.2619	0.9748	1.295	—	—	—	0.1604	—
2300	1.2669	0.9798	1.293	—	—	—	0.1535	—
2400	1.2717	0.9846	1.292	—	—	—	0.1471	—
2500	1.2762	0.9891	1.290	—	—	—	0.1412	—
2600	1.2803	0.9932	1.289	—	—	—	0.1358	—
2700	1.2843	0.9972	1.288	—	—	—	0.1307	—
2800	1.2881	1.0010	1.287	—	—	—	0.1261	—
2900	1.2916	1.0045	1.286	—	—	—	0.1217	—
3000	1.2949	1.0078	1.285	—	—	—	0.1177	—

The values for air can also be used with reasonable accuracy for CO, N₂ and O₂.

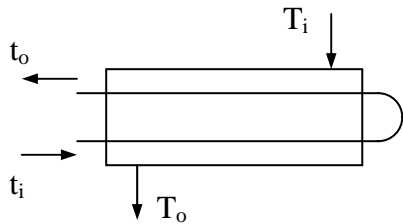
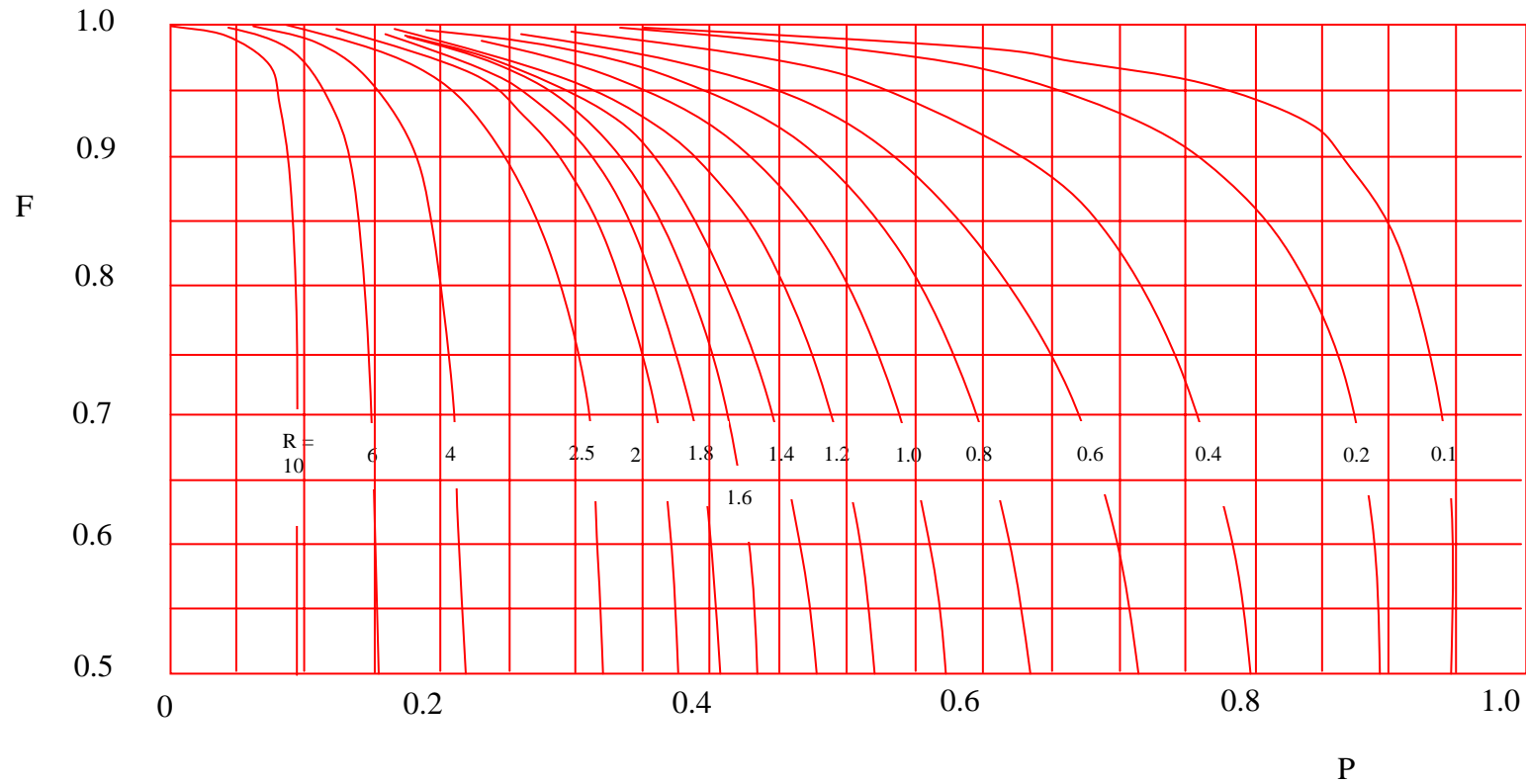
The values of the thermodynamic properties c_v and c_p on pp. 16 and 17 are those at zero pressure. The values for the gases are quite accurate over a wide range of pressure, but those for the vapours increase appreciably with pressure.

The transport properties μ and k for air are accurate over a wide range of pressure, except at such low pressures that the mean free path of the molecules is comparable to the distance between the solid surfaces containing the gas.

At high temperatures (>1500 K for air) dissociation becomes appreciable and pressure is a significant variable for both gases and vapours: the values on pp. 16 and 17 apply only to undissociated states.



Effectiveness of a shell-and-tube heat exchanger with one shell and any multiple of two tube passes (2, 4, etc)



$$P = \frac{t_o - t_i}{T_i - t_i}$$

$$R = \frac{T_i - T_o}{t_o - t_i}$$

Correction factor for a shell-and-tube heat exchanger with one shell and any multiple of two tube passes (two, four, etc tube passes)

Generalised Solution Methodologies

If a surface is open then any radiation that passes through it is unlikely to return. Therefore an open surface acts as a black body with $\varepsilon = 1$.

If a surface is adiabatic then the resistance $(1 - \varepsilon) / \varepsilon A$ becomes irrelevant and $E_{bi} = J_i$

If one surface cannot see another then the view factor F will be zero and the resistance will be infinite. This indicates there will be no heat transfer between those surfaces by radiation.

For general problems involving more than 3 bodies a large number of simultaneous linear equations may need to be solved and this can be done by Gauss-Siedel method. The general set of equations are given by:

$$[M][J] = [S]$$

where $[M]$ is a $n \times n$ matrix for n bodies where the components are given by:

$$M_{ij=i} = \frac{1 - (1 - \varepsilon_i)F_{i-i}}{\varepsilon_i} \quad \& \quad M_{ij \neq i} = \frac{-(1 - \varepsilon_i)F_{i-j}}{\varepsilon_i}$$

$[J]$ and $[S]$ are $1 \times n$ matrices which are given by:

$$[J] = \begin{bmatrix} J_1 \\ J_2 \\ \dots \\ J_n \end{bmatrix} \quad \& \quad [S] = \begin{bmatrix} \sigma T_1^4 \\ \sigma T_2^4 \\ \dots \\ \sigma T_n^4 \end{bmatrix}$$

For some problem it is easier to write [S] matrix as:

$$[S] = \begin{bmatrix} \sigma.T_1^4 \\ \sigma.T_2^4 \\ \vdots \\ \sigma.T_k^4 \\ q_{k+1} \\ \dots \\ \dots \\ q_n \end{bmatrix}$$

For this case [M] matrix takes the following form:

$$M_{ij=i} = \frac{1 - (1 - \varepsilon_i)F_{i-i}}{\varepsilon_i} \quad \& \quad M_{ij \neq i} = \frac{-(1 - \varepsilon_i)F_{i-j}}{\varepsilon_i} \quad \text{for } i = 1, 2, \dots, k$$

$$M_{ij=i} = 1 - F_{i-i} \quad \& \quad M_{ij \neq i} = -F_{i-j} \quad \text{for } i = k+1, k+2, \dots, n$$

The radiosity matrix [J] remains unchanged and one needs to solve [M] [J] = [S]